

# 1 Nonlinear stresses and temperatures in transient adiabatic and shear flows via nonequilibrium 2 molecular dynamics: Three definitions of temperature

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6 We compare nonlinear stresses and temperatures for adiabatic-shear flows, using up to 262 144 particles,  
7 with those from corresponding homogeneous and inhomogeneous flows. Two varieties of kinetic temperature  
8 tensors are compared to the configurational temperatures. This comparison of temperatures led us to two  
9 findings beyond our original goal of analyzing shear algorithms. First, we found an improved form for local  
10 instantaneous velocity fluctuations, as calculated with smooth-particle weighting functions. Second, we came  
11 upon the previously unrecognized contribution of rotation to the configurational temperature.

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## 14 I. INTRODUCTION

15 Hoover *et al.* [1] studied the nonlinear stresses and tem-  
16 perature changes induced by shear in a variety of stationary  
17 flows. They used nonequilibrium molecular dynamics to  
18 compare several popular algorithms. The algorithms are de-  
19 scribed briefly in Sec. II. See Figs. 1 and 2 for the geometries  
20 used to induce the flows. This work used a smooth repulsive  
21 soft-sphere potential [2] with a range of unity,

$$22 \quad \phi(r < 1) = 100(1 - r^2)^4.$$

23 The particle mass, energy per particle, and density were all  
24 chosen equal to unity. These conditions correspond to a  
25 dense fluid at about 2/3 the freezing pressure,

$$26 \quad m = 1; \quad E/N = (K + \Phi)/N = 1; \quad \rho = Nm/V = 1.$$

27 Thermostat or ergostat forces were used to generate station-  
28 ary states. There, as well as in the present work, we choose  $x$   
29 for the flow direction and  $\dot{\epsilon} = 0.5$  for the strain rate, where the  
30 time-averaged velocity component of the flow  $v_x$  increases  
31 linearly in the  $y$  direction,

$$32 \quad \langle v_x(y) \rangle = \dot{\epsilon}y.$$

33 Three-dimensional homogeneous periodic simulations (Fig.  
34 1 shows a two-dimensional version) gave

$$35 \quad T_{yy} > T_{xx} > T_{zz}, \quad P_{yy} > P_{xx} > P_{zz} \text{ [Doll's algorithm]},$$

$$36 \quad T_{xx} > T_{yy} > T_{zz}, \quad P_{xx} > P_{yy} > P_{zz} \text{ [Sllod algorithm]},$$

37 and differed qualitatively from the corresponding three-  
38 dimensional boundary-driven results (Fig. 2 shows a two-  
39 dimensional version),

$$40 \quad T_{xx} > T_{zz} > T_{yy}, \quad P_{xx} > P_{zz} > P_{yy} \text{ [boundary-driven]}.$$

41 The main conclusion drawn from that work was that nei-  
42 ther homogeneous method, Doll's [3] nor Sllod [4], success-  
43 fully reproduces the more-physical boundary-driven results  
44 [5]. To quote Ref. [1] "the Doll's and Sllod algorithms pre-  
45 dict opposite signs for this normal-stress difference [ $P_{xx}$   
46  $-P_{yy}$ ], with the Sllod approach definitely wrong, but some-  
47 what closer to the (boundary-driven) truth."

48 Evans [6] objected to this conclusion, stating that the  
49 Sllod algorithm is "exact." He is of course correct, in the

sense that the Sllod algorithm is nothing more than Newton's  
equations of motion written in a different coordinate frame, a  
Lagrangian frame moving along with the sheared fluid. But  
because Newton's equations by themselves cannot lead to  
nonequilibrium steady states, the Sllod algorithm is exact in  
the rather limited case of isolated systems.

The misunderstanding evident in Evans' remark led us to  
undertake the present work. Instead of considering steady  
states, which seem to us the simplest situation, Evans had in  
mind a time-dependent spatially periodic adiabatic deformation.  
When no thermostat or ergostat forces are used in the  
equations of motion, the shear deformation is adiabatic with  
continuous heating. In the adiabatic case no steady state re-  
sults and the Sllod equations of motion are equivalent to  
Newton's equations of motion for a system undergoing peri-  
odic deformation with strain rate  $\dot{\epsilon}$ . Just as in the thermo-  
stated case the normal stresses and temperatures differ. Not  
only the magnitudes but also the orderings of these compo-  
nents can, and do, differ from those found in steady states.

In this paper we motivate and describe large-scale  
adiabatic-shear simulations and discuss the interpretation of  
these simulations. These simulations use periodic boundary

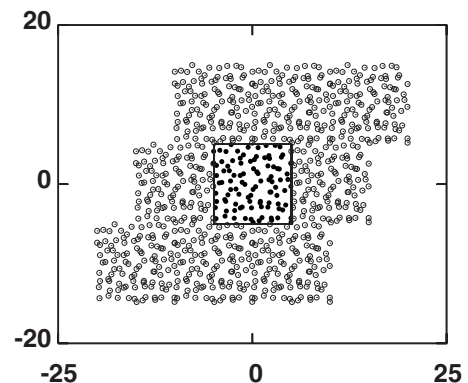


FIG. 1. Two-dimensional version of periodic homogeneous isoenergetic shear flow. Eight periodic images of the central  $N$ -particle system are shown. In this figure, as in all the others, the units are dimensionless, where the volume per particle, the particle mass, and Boltzmann's constant  $k$  are all equal to unity. Thus the plotted lengths, times, pressures, and temperatures are all dimensionless.

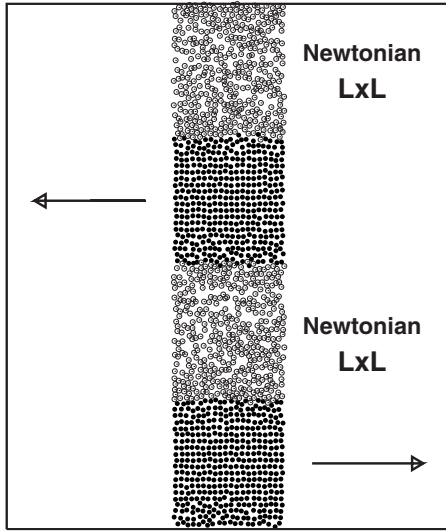


FIG. 2. Two-dimensional version of periodic inhomogeneous boundary-driven shear flow. The system consists of four separate chambers of  $N$  particles each. The chambers indicated by arrows are driven to the right and left by moving tether forces. Heat is extracted from the driven chambers to maintain constant internal energy there. The boundary-driven motion of the other two chambers is purely Newtonian.

72 conditions, just as shown in Fig. 1, but are extended here to  
 73 three Cartesian space dimensions  $\{x, y, z\}$ . We take into ac-  
 74 count the important role of fluctuations in defining local val-  
 75 ues of the velocity, and the temperature and stress tensors.  
 76 Section II describes the algorithms, and Sec. III describes the  
 77 various definitions of temperature for nonequilibrium (as  
 78 well as equilibrium) systems. Section IV outlines the results  
 79 of the current simulations. Section V gives the conclusions  
 80 we have reached as a result of this work. Section VI suggests  
 81 extensions of this work.

## 82 II. SHEAR FLOW ALGORITHMS

83 Two numerical algorithms, “Sllod” and “Doll’s,” for spa-  
 84 tially periodic shear flow in a volume  $V$  both satisfy the  
 85 macroscopic energy-balance relation  $\dot{E} = -\dot{\epsilon} P_{xy} V$ . The corre-  
 86 sponding solutions differ in effects of order  $\dot{\epsilon}^2$ , with  $P_{xx}$   
 87  $> P_{yy}$  in the Sllod case and the reverse using Doll’s algo-  
 88 rithm. To describe nonequilibrium situations it is natural to  
 89 introduce the gradients and time derivatives of these vari-  
 90 ables, with the simplest situations those “stationary states”  
 91 (necessarily driven by external forces or heat sources) in  
 92 which all the partial time derivatives (the rates of change at a  
 93 fixed location) vanish. Steady shear flow can be simulated  
 94 with homogeneous sources and sinks of momentum and en-  
 95 ergy through the Doll’s and Sllod algorithms. The *adiabatic*  
 96 versions of these equations of motion (no thermostats or er-  
 97 gostats) introduce an overall flow field imposed with the pa-  
 98 rameter  $\dot{\epsilon}$  through periodic boundary conditions,

$$99 \quad \dot{x} \equiv (p_x/m) + \dot{\epsilon}y; \quad \dot{y} = (p_y/m) \quad [\text{Sllod or Doll's}].$$

100 The Sllod algorithm is simply a rewriting of Newton’s  
 101 equations of motion,

$$\ddot{x} = (F_x/m); \quad \ddot{y} = (F_y/m). \quad 102$$

To see this we introduce the new “momenta”  $p_x$  and  $p_y$ , 103

$$p_x \equiv m(\dot{x} - \dot{\epsilon}y); \quad p_y \equiv m\dot{y}. \quad 104$$

The time derivative of these definitions then gives the Sllod 105  
 algorithm for  $\{\dot{x}, \dot{y}, \dot{p}_x, \dot{p}_y\}$ , 106

$$\dot{x} = (p_x/m) + \dot{\epsilon}y; \quad \dot{y} = (p_y/m); \quad 107$$

$$\dot{p}_x \equiv F_x - m\dot{\epsilon}\dot{y} = F_x - \dot{\epsilon}p_y; \quad \dot{p}_y \equiv F_y. \quad 108$$

In the laboratory frame (where one sees the overall strain rate 109  
 $\dot{\epsilon}y$  induced by the periodic boundary conditions)  $p_x$  and  $p_y$  110  
 are just  $m\dot{x}$  and  $m\dot{y}$ , and the motion follows from the usual 111  
 Hamiltonian, 112

$$\mathcal{H}_{\text{Lab}} = \sum p^2/(2m) + \Phi. \quad 113$$

If, as in the Sllod algorithm, the momentum  $(p_x, p_y)$  is 114  
 defined instead in the comoving frame then there is no analo- 115  
 gous Hamiltonian. To see this in detail suppose that the co- 116  
 moving equations of motion (describing the Newtonian dyn- 117  
 amics) could be derived from a hypothetical comoving 118  
 Hamiltonian,  $\mathcal{H}_{\text{com}}(\{x, y, p_x, p_y\})$ , 119

$$\dot{x} = +(\partial\mathcal{H}_{\text{com}}/\partial p_x) = (p_x/m) + \dot{\epsilon}y, \quad 120$$

$$\dot{y} = +(\partial\mathcal{H}_{\text{com}}/\partial p_y) = (p_y/m), \quad 121$$

$$\dot{p}_x = -(\partial\mathcal{H}_{\text{com}}/\partial x) = F_x - \dot{\epsilon}p_y, \quad \dot{p}_y = -(\partial\mathcal{H}_{\text{com}}/\partial y) = F_y. \quad 122$$

The second partial derivatives of the hypothetical Hamil- 123  
 tonian with respect to  $y$  and  $p_x$  should be equal. But we find 124  
 instead 125

$$(\partial/\partial y)(\partial\mathcal{H}_{\text{com}}/\partial p_x) = (\partial/\partial y)(\dot{\epsilon}y) = \dot{\epsilon} \quad 126$$

and 127

$$(\partial/\partial p_x)(\partial\mathcal{H}_{\text{com}}/\partial y) = (\partial/\partial p_x)(-F_y) = 0, \quad 128$$

showing that there is no such comoving Hamiltonian. 129

On the other hand, the very similar Doll’s-Tensor equa- 130  
 tions of motion (which are not Newtonian) do follow from a 131  
 special Hamiltonian appropriate to the comoving frame, 132

$$\mathcal{H}_{\text{Doll's}} = \sum p^2/(2m) + \Phi + \dot{\epsilon} \sum y p_x; \quad 133$$

$$+ (\partial\mathcal{H}_{\text{Doll's}}/\partial p_x) = \dot{x} = (p_x/m) + \dot{\epsilon}y; \quad 134$$

$$+ (\partial\mathcal{H}_{\text{Doll's}}/\partial p_y) = \dot{y} = (p_y/m). \quad 135$$

$$- (\partial\mathcal{H}_{\text{Doll's}}/\partial x) = \dot{p}_x = F_x; \quad - (\partial\mathcal{H}_{\text{Doll's}}/\partial y) = \dot{p}_y = F_y - \dot{\epsilon}p_x. \quad 136$$

Both the foregoing Sllod and Doll’s sets of motion equa- 137  
 tions are adiabatic, so that the systems they describe heat due 138  
 to viscous shear as time goes on. Additional time-reversible 139  
 frictional forces of the form  $-\zeta p$  can be added to either set of 140  
 motion equations to keep the energy or the temperature con- 141  
 stant [7,8], 142

143  $\{\Delta F = -\zeta p; p_x \equiv m(\dot{x} - \dot{\epsilon}y); p_y = m\dot{y}; p_z = m\dot{z}\}.$

144 The frictional forces make it possible to explore a spatially  
145 homogeneous nonequilibrium steady state with definite val-  
146 ues of the (time-averaged) stress and temperature. Both these  
147 nonequilibrium properties need proper definitions. We con-  
148 sider several alternative definitions of temperature in Sec. III.

### 149 III. DEFINITIONS OF TEMPERATURE

150 In statistical mechanics a longstanding definition of tem-  
151 perature has been kinetic based on the physical picture of an  
152 ideal-gas thermometer [7,9]. Measuring the momenta  $\{p\}$   
153 relative to the comoving frame of the kinetic thermometer,  
154 the kinetic-theory definition is

155  $kT_{xx} \equiv \langle p_x^2/m \rangle; kT_{yy} \equiv \langle p_y^2/m \rangle; kT_{zz} \equiv \langle p_z^2/m \rangle.$

156 A simple mechanical model capable of measuring all three  
157 temperatures simultaneously is a dilute gas of parallel hard  
158 cubes [10].

159 There is also a configurational analog [11,12],

160  $kT_{xx} = \langle F_x^2 \rangle / \langle \nabla_x^2 \mathcal{H} \rangle; kT_{yy} = \langle F_y^2 \rangle / \langle \nabla_y^2 \mathcal{H} \rangle;$

161  $kT_{zz} = \langle F_z^2 \rangle / \langle \nabla_z^2 \mathcal{H} \rangle.$

162 The configurational temperature has no clear connection to a  
163 physical model of a thermometer but follows instead [11]  
164 from a formal integration by parts of the canonical average  
165 of  $\nabla^2 \mathcal{H}$ ,

166  $\langle \nabla^2 \mathcal{H} \rangle \equiv \langle (\nabla \mathcal{H})^2 / kT \rangle.$

167 Both temperature definitions, kinetic and configurational,  
168 have associated ambiguities [1]: fluctuations in the kinetic  
169 case and [2] rotation in the configurational case. Consider  
170 fluctuations first. The local velocity fluctuates in time. The  
171 thermal momentum in the kinetic definition has to be mea-  
172 sured in a “comoving” frame. Once the velocity is a local  
173 quantity, as well as a time-dependent quantity, its definition  
174 becomes crucial. Here we adopt a modification of the  
175 “smooth-particle” definition of local velocity [13],

176  $v(r) \equiv \sum w(|r - r_i| < h) v_i / \sum w(|r - r_i| < h),$

177 where  $v_i$  is the velocity of particle  $i$  and that particle lies  
178 within the range  $h$  of the smooth-particle weighting function  
179  $w(r < h)$ .

180 Smooth-particle applied mechanics (SPAM) [13] provides  
181 spatially very smooth material properties (such as density,  
182 velocity, stress, and energy) with two continuous spatial de-  
183 rivatives. The definitions of these properties require a smooth  
184 weighting function,  $w(r < h)$ , which must be continuously  
185 twice differentiable, normalized, and which must also have a  
186 finite range  $h$ . Here we adopt the simplest such weighting  
187 function meeting these requirements, Lucy’s. In three dimen-  
188 sions Lucy’s form for  $w$  is

189  $w_{\text{Lucy}} = \frac{105}{16\pi h^3} [1 - 6x^2 + 8x^3 - 3x^4]; x \equiv |r|/h;$

$$\rightarrow \int_0^\infty 4\pi r^2 w(r < h) dr \equiv \int_0^h 4\pi r^2 w(r < h) dr \equiv 1. \quad 190$$

In Sec. IV we show that the smooth particulate velocity fluctua-  
191 tions measured as temperature are best defined through a  
192 slight modification of the smooth-particle approach, in which  
193 the “self” contributions to the particle sums,  $\Sigma w$  and  $\Sigma wv$ ,  
194 are absent. This modification reduces the number depen-  
195 dence inherent in comparing atomistic simulations to con-  
196 tinuum predictions. 197

At first sight, the configurational definition of temperature  
198 has an advantage over the kinetic one that a calculation of  
199 the stream velocity is not required. But the current work led  
200 us to recognize a difficulty in defining configurational tem-  
201 perature away from equilibrium. Consider rotation. Particu-  
202 larly in turbulent flows, *rotation* is important. Although con-  
203 figurational temperature has been touted as a way to avoid  
204 defining a local velocity [13], it also contains a small and  
205 subtle ambiguity—configurational temperature depends on  
206 rotation rate. 207

A rotating rigid body generates centrifugal forces of order  
208  $\omega^2 r$  (offset by tensile forces) at a distance  $r$  from the center  
209 of mass. The tensile forces contribute to the configurational  
210 temperature definition, 211

$$kT_C \equiv \langle F^2 \rangle / \langle \nabla^2 \mathcal{H} \rangle, \quad 212$$

while the centrifugal ones do not, so that perimeter particles  
213 are apparently “hotter” than the cooler interior by (relatively  
214 small) contributions of order  $\omega^4$ . 215

To see this in a simple two-dimensional example, consider  
216 the point  $(x, y)$  viewed from an  $(X, Y)$  coordinate system ro-  
217 tating counterclockwise at the angular frequency  $\omega$ , 218

$$X = x \cos(\omega t) + y \sin(\omega t); Y = y \cos(\omega t) - x \sin(\omega t). \quad 219$$

Two time differentiations, evaluated at time  $t=0$ , give the  
220 Coriolis, centrifugal (rotating-frame), and centripetal  
221 (laboratory-frame) forces, 222

$$\ddot{X} = \ddot{x} + 2\omega\dot{y} - \omega^2 x = \ddot{x} + 2\omega\dot{Y} + \omega^2 X, \quad 223$$

$$\ddot{Y} = \ddot{y} - 2\omega\dot{x} - \omega^2 y = \ddot{y} - 2\omega\dot{X} + \omega^2 Y. \quad 224$$

For rigid rotation at an angular velocity  $\omega$  a particle at  
225  $(x, y) = (r, 0)$  with laboratory-frame velocity  $(0, \omega r)$  and ac-  
226 celeration  $(F_x, 0)/m = (-\omega^2 r, 0)$ , the  $\ddot{X}$  equation becomes 227

$$\ddot{X} = \ddot{x} + 2\omega\dot{y} - \omega^2 x = (F_x/m) + 2\omega^2 r - \omega^2 r \equiv 0 = (F_x/m) + \omega^2 r, \quad 228$$

showing that the atomistic force  $(F_x/m)$  exactly offsets the  
229 centrifugal force  $\omega^2 r$ . Thus the configurational temperature  
230 for rigid rotation is proportional to  $r^2 \omega^4$ . 231

The numerical work described in Sec. IV, for relatively  
232 gentle shear flows, supports the view that kinetic temperature  
233 is both simpler and better behaved than configurational tem-  
234 perature, with smaller fluctuations in both space and time. In  
235 a work still in progress we contrast the two approaches for  
236 the problem of a strong shockwave. 237

**238** IV. SIMULATIONS AND RESULTS

**239** Throughout, we focus on a dense fluid in three space di-  
**240** mensions, with the short-ranged “soft-sphere” potential of  
**241** Refs. [1,2],

$$\phi(r < 1) = 100(1 - r^2)^4, \quad \Phi = \sum_{i < j} \phi(|r_{ij}|),$$

**243** where the sum over pairs includes all particle pairs within a  
**244** distance unity. The total energy of the system consists of a  
**245** kinetic part in addition to the potential energy  $\Phi$ ,

$$E = K + \Phi, \quad K = \sum p^2/(2m).$$

**247** We focus on the dense-fluid state of Ref. [1], with a density  
**248** and energy per particle of unity,

$$E/N = Nm/V = N/V = 1,$$

$$6 \times 10^3 = 216 \leq N \leq 2744 = 14 \times 10^3.$$

**251** Data for homogeneous isoenergetic Doll’s and Sllod  
**252** simulations are given in Ref. [1]] along with complementary  
**253** results for boundary-thermostated flows. That study showed  
**254** that the normal-stress differences in the homogeneous simu-  
**255** lations are very different from those found in boundary-  
**256** driven flows. The number dependence in the temperatures  
**257** and normal-stress differences of the homogeneous flows is  
**258** nearly negligible, no more than  $1/N$  once the number of  
**259** particles  $N$  is a few hundred. By contrast, the boundary-  
**260** driven temperatures are quite different, as the midstream  
**261** temperature increases as  $N^{2/3}$ .

**262** Here we consider in addition *adiabatic deformation* with  
**263** the Newtonian motion driven by shearing boundary condi-  
**264** tions and without any thermostat or ergostat forces. The ini-  
**265** tial state is a cubic lattice with a kinetic temperature of  $kT$   
**266**  $= 0.01$  and an initial energy per particle of  $E = K + \Phi$   
**267**  $= 0.015N + 0$  (because the nearest-neighbor separation is ini-  
**268** tially unity just beyond the range of the repulsive forces). We  
**269** compute and compare two different kinetic temperatures,  
**270** each with the three components  $\{T_{xx}, T_{yy}, T_{zz}\}$ . The *time-*  
**271** *averaged temperature*,  $kT^{\text{TA}}$  is

$$kT_{xx}^{\text{TA}} \equiv \langle m(\dot{x}_i - \dot{\epsilon}_y)^2 \rangle,$$

$$kT_{yy}^{\text{TA}} \equiv \langle m\dot{y}^2 \rangle,$$

$$kT_{zz}^{\text{TA}} \equiv \langle m\dot{z}^2 \rangle,$$

**275** while the instantaneous temperature,  $kT^{\text{inst}}$ , is

$$kT_{xx}^{\text{inst}} \equiv \langle m[\dot{x}_i - v_x(r_i, t)]^2 \rangle,$$

$$kT_{yy}^{\text{inst}} \equiv \langle m[\dot{y}_i - v_y(r_i, t)]^2 \rangle,$$

$$kT_{zz}^{\text{inst}} \equiv \langle m[\dot{z}_i - v_z(r_i, t)]^2 \rangle,$$

**279** where the instantaneous velocity at particle  $i$ ’s location is a  
**280** modified version of the usual smooth-particle average [13],

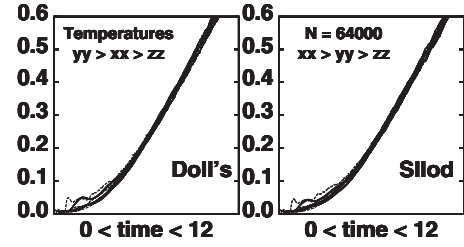


FIG. 3. Overall adiabatic temperature variations for adiabatic-shear flows with  $64 \times 64 \times 64$  soft spheres with an initial kinetic temperature of 0.01. The strain rate  $du_x/dy = \dot{\epsilon}$  is 0.5.  $\{T_{xx}, T_{yy}, T_{zz}\}$  are plotted here. For times greater than 3 neither algorithm shows significant differences between the temperatures on the scale of the plots.

$$v(r_i, t) = \sum_{j \neq i} w(r_{ij}) v_j / \sum_{j \neq i} w(r_{ij}).$$

For simplicity we choose Lucy’s weight function with the  
 range  $h=3$ ,

$$w(r < h) = \frac{105}{16\pi h^3} (1 - 6x^2 + 8x^3 - 3x^4), \quad x \equiv r/h,$$

for the evaluation of all the smooth-particle sums.

In smooth-particle simulations [13] the “self-terms,” in  
 the two particle sums,  $w_{ii}v_i$  and  $w_{ii}$  are always included. In  
 analyzing equilibrium molecular-dynamics simulations for  
 local velocity fluctuations (the usual kinetic temperature),  
 numerical work shows that there is a much better correspon-  
 dence with equilibrium temperature when the self-terms are  
 omitted. We have followed that practice here. If the self-  
 terms are included in computing the local stream velocity the  
 resulting kinetic temperatures are roughly 10% lower. With  
 the self-terms omitted the three temperature definitions,  
 time-averaged kinetic, instantaneous kinetic, and configura-  
 tional, all give similar results.

The need for excluding the “self”-contributions can be  
 rationalized by considering an equilibrium particle  $i$  at loca-  
 tion  $r_i$  with velocity  $v_i$ . With its neighbor velocities uncorre-  
 lated (as they are at equilibrium within terms of order  $1/N$ ),  
 the smooth-particle velocity at  $r_i$  is, on average,

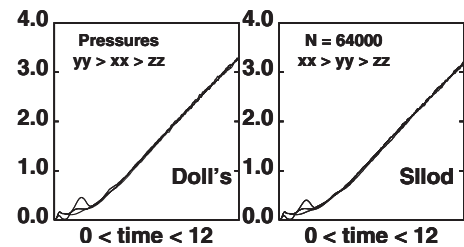


FIG. 4. Overall adiabatic pressure variation for adiabatic-shear flows with  $64 \times 64 \times 64$  soft spheres with an initial kinetic temperature of 0.01. The strain rate  $du_x/dy = \dot{\epsilon}$  is 0.5.  $\{P_{xx}, P_{yy}, P_{zz}\}$  are plotted here. For times greater than 3 neither algorithm shows significant differences between the pressures on the scale of the plots.

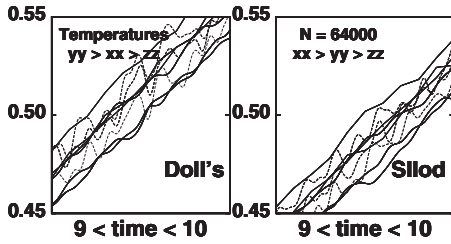


FIG. 5.  $\{T_{xx}, T_{yy}, T_{zz}\}$  are plotted here for portions of 64 000-particle adiabatic simulations of Figs. 3 and 4. The heaviest lines show the laboratory-frame kinetic temperature; the medium lines show kinetic temperature relative to the instantaneous smooth-particle velocity. The light dashed lines show the configurational temperatures, which fluctuate more wildly than the kinetic temperatures. In the steady-state simulations of Ref. [1] the Doll's kinetic temperatures  $\{0.496, 0.508, 0.493\}$  and the Sllod kinetic temperatures  $\{0.507, 0.497, 0.493\}$  correspond to an internal per particle energy of exactly unity.

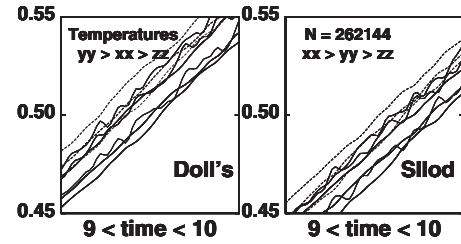


FIG. 7.  $\{T_{xx}, T_{yy}, T_{zz}\}$  are plotted here for portions of 262 144-particle adiabatic-shear simulations. The heaviest lines show the laboratory-frame kinetic temperature; the medium lines show kinetic temperature relative to the instantaneous smooth-particle velocity. The light dashed lines show the configurational temperatures, which fluctuate more than the kinetic temperatures. The kinetic temperatures are nearly the same as those in the stationary shear simulations of Ref. [1].

$$v_{\text{SPAM}}(r_i) \equiv \sum_j w_{ij} v_j / \sum_j w_{ij} \approx w(0) v_i / n, \quad n \equiv N/V,$$

so that the temperature, based on the velocity fluctuations as measured by the differences,  $\{v_i - \langle v_{\text{SPAM}}(r_i) \rangle\}$ , is reduced by a factor of  $[1 - w(0)/n]^2$ . In the present work the number density  $n$  is unity. In accord with this equilibrium argument, we have excluded the “self-terms” in the kinetic parts of the temperatures and pressures illustrated in the figures.

Figures 3 and 4 show the overall increase in temperature and pressure beginning with a homogeneous cubic crystal, at a kinetic temperature of 0.01, and ending at a homogeneous shearing fluid state with a temperature somewhat greater than 0.5. The details of the temperature and pressure for 64 000 particles, in the vicinity of  $kT \approx 0.5$ , are shown in Figs. 5 and 6. The fluctuations in the data can be reduced by using even larger systems. Compare Figs. 5 and 6 with the corresponding results for 262 144 particles, shown in Figs. 7 and 8. In these latter simulations the internal energy per particle reaches unity for the Sllod algorithm at a time of 9.799 and for the Doll's algorithm at a time of 9.480.

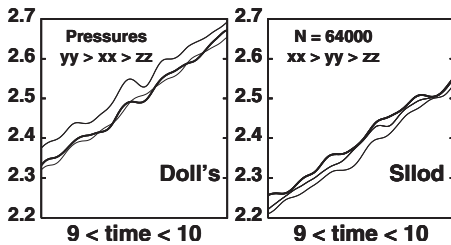


FIG. 6.  $\{P_{xx}, P_{yy}, P_{zz}\}$  are plotted here for portions of the adiabatic simulations of Figs. 3 and 4 (using laboratory-frame kinetic contributions) and correspond to the heavy, medium, and light lines, respectively. In the steady-state simulations of Ref. [1] the Doll's pressures (using laboratory-frame kinetic parts)  $\{2.496, 2.528, 2.482\}$  and the Sllod pressures  $\{2.516, 2.509, 2.484\}$  correspond to an internal per particle energy of exactly unity and an average temperature of about 0.5. The shear stress is about the same for the two algorithms,  $\sigma_{xy} = 0.343$ .

At a fixed strain rate of 0.5, small-system fluctuations can completely obscure the orderings of  $\{T_{ii}\}$  and  $\{P_{ii}\}$ . By increasing the system size it is possible to verify that the transient fluctuating temperatures and stresses in adiabatic deformation are close to those of the isoenergetic periodic shears, with the orderings  $y > x > z$  for Doll's and  $x > y > z$  for Sllod. Neither algorithm reproduces the boundary-driven ordering (at the same density, strain rate, and energy)  $x > z > y$ . We discuss this finding in Sec. V.

V. NONEQUILIBRIUM CONSTITUTIVE RELATIONS

Models for continuum mechanics follow from conservation of mass, momentum, and energy. The differential expressions of these conservation relations are the continuity equation, the equation of motion (which introduces the pressure tensor  $P$  as the comoving momentum flux), and the energy equation (which introduces the heat flux vector  $Q$  as the comoving energy flux),

$$\dot{\rho} = -\rho \nabla \cdot v; \quad \rho \dot{v} = -\nabla \cdot P; \quad \rho \dot{e} = -\nabla v : P - \nabla \cdot Q.$$

The time-and-space-dependent state variables of hydrodynamics are taken from equilibrium thermodynamics, extended to the case in which gradients and time dependence can occur. The state variables at location  $r$  and time  $t$  are the density, velocity, and energy,  $\{\rho(r, t), v(r, t), e(r, t)\}$ , and it is assumed that the pressure and heat flux can be defined in

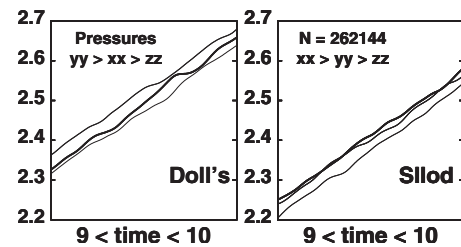


FIG. 8.  $\{P_{xx}, P_{yy}, P_{zz}\}$  are plotted here for portions of 262 144-particle adiabatic-shear simulations and correspond to the heavy, medium, and light lines, respectively. The configurational parts of the pressure are slightly, but significantly, larger than those found in the stationary shear simulations of Ref. [1].

346 terms of the present values, the gradients, and possibly the  
347 past histories of these same variables.

348 In the present work we have seen that both the tempera-  
349 ture (extended from the scalar thermodynamic variable to  
350 tensor values) and the stress can differ for two systems with  
351 identical densities, strain rates, energies, and constitutive re-  
352 lations (because the underlying particles are the same). Evi-  
353 dently both temperature and stress depend on additional state  
354 variables. The relative independence of the normal-stress dif-  
355 ferences to the system size [1]  $L$  suggests that the discrep-  
356 ancy between periodic and boundary-driven systems is in-  
357 sensitive to second derivatives,  $\{\nabla\nabla\rho, \nabla\nabla v, \nabla\nabla e\}$ , all of  
358 which vary as  $L^{-2}$ . From the constitutive standpoint it is sim-  
359 plest to imagine a dependence of the normal-stress differ-  
360 ences and the temperature tensor on the rate of heating  $\dot{e}$ .  
361 Such a dependence could be used to describe the deviation of  
362 the adiabatic transient flows from the corresponding station-  
363 ary flows. Finding an additional independent constitutive  
364 variable to distinguish stationary boundary-driven flows  
365 from stationary homogeneous flows is a challenging research  
366 goal.

## 367 VI. SUMMARY

368 Evans' emphasis on the exactness of the Sllod algorithm  
369 (restricted to adiabatic flows) is confirmed here, as Sllod is  
370 nothing but Newton in a different coordinate frame. But it  
371 must be noted that the large-system adiabatic pressure tensor  
372 exhibits clear differences from the stationary pressure tensor  
373 at the same energy, density, and strain rate. Sufficiently large  
374 systems, with hundreds of thousands of particles, show that  
375 the nonequilibrium temperature tensors of adiabatic transient  
376 flows are very similar to those of homogeneous periodic sta-  
377 tionary flows. The "realism" of the adiabatic flows is ques-  
378 tionable because real boundaries, which normally drive, con-  
379 strain, and cool flows, are absent. The diffusion time for an  
380  $N$ -particle system driven by a strain rate incorporated in its  
381 periodic boundary conditions varies as  $N^{2/3}$ , so that simula-  
382 tion results depend increasingly on their initial conditions as  
383 system size increases.

384 An interesting finding of the present work is that the  
385 smooth-particle calculation of local velocity (when that ve-  
386 locity is needed for the computation of the local tempera-  
387 ture),  $v_{\text{SPAM}} = \sum wv / \sum w$ , is best modified by omitting the self-  
388 terms in both sums. In a motionless equilibrium system the  
389 "correct" average velocity, about which thermal fluctuations  
390 are measured, should vanish. The usual smooth-particle av-  
391 erage at particle  $i$ ,  $\langle v_{\text{SPAM}} \rangle$ , includes a contribution of order  
392  $v_i$  divided by the number of particles included in the sums.

For typical useful choices of the weight function and its 393  
range  $h$ , field variables, such as density and velocity, include 394  
contributions from a few dozen particles. When temperature 395  
is estimated from fluctuations about a smooth-particle aver- 396  
age, omitting the self-terms gives an improved temperature 397  
estimate. The improvement is of the order of 10%. 398

From the constitutive standpoint it is simplest to "explain" 399  
the difference between the adiabatic and boundary-driven 400  
nonlinear properties through a dependence on  $\dot{e}$ , where  $e$  is 401  
the internal energy per unit mass. Though the configurational 402  
temperature tensor avoids the problem of defining a local 403  
stream velocity, it still includes the effect of rotational con- 404  
tributions, giving rise to "temperature gradients" based on 405  
centrifugal forces in the absence of heat flow. Fortunately 406  
these rotational temperature contributions are small, of order 407  
 $\dot{e}^4$ . 408

## VII. WHAT TO DO? 409

One referee asked us Lenin's famous question with regard 410  
to this work's consequences. We hope to stimulate further 411  
investigations of microscopic systems from macroscopic 412  
points of view. The microscopic analogs of macroscopic 413  
temperature, stress, and fluctuations are imperfect but vital in 414  
drawing macroscopic conclusions from particle simulations. 415  
There is much to do in understanding this correspondence 416  
better for more complex systems with rotational and vibra- 417  
tional degrees of freedom. Simulations and theories of elon- 418  
gational flow have led to unresolved controversies as to the 419  
"right way" to simulate such flows. See, for instance Refs. 420  
[28–36] cited in our Ref. [1]. We believe that the nonlinear 421  
aspects of steady deformational flows deserve more study. 422  
For unsteady flows even an "exact" algorithm such as Sllod 423  
depends on an essential way on the initial conditions unless 424  
the deformation rate is very small. 425

Shockwaves provide more extreme tests of the correspon- 426  
dence between microscopic and macroscopic models. The 427  
significance of temperature for quantum systems away from 428  
equilibrium needs elucidation too. We are confident that 429  
progress along all of these lines can best be achieved by 430  
carrying out, analyzing, and comparing series of simulations 431  
such as those described in the present work. 432

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